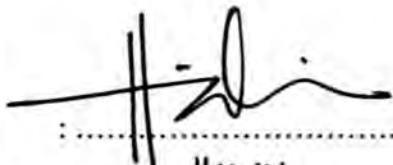


APPROVAL

"I hereby declare that I have read this thesis and in my opinion this thesis is sufficient in term of scope and quality for the award of degree of Bachelor Mechanical Engineering (Automotive)"

Signature



Supervisor's Name

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DISC BRAKE PERFORMANCE: FEA

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A thesis report submitted in partial
fulfillment of the requirements for the award of the
Degree of Bachelor Mechanical Engineering (Automotive)

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May 2008

DECLARATION

"I hereby declared that this thesis is result of my own research except as cited in the references"

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Date :*13 - 05 - 2008*.....

DEDICATION

*Special dedicate to my beloved parent, family, supervisor, friends and all that me to
finish my thesis.*

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ABSTRAK

Brek cakera atau rotor menyediakan permukaan membrek atau dikenali sebagai permukaan geseran bagi brek pad untuk mengosok padanya apabila brek dikenakan. Fungsi utama brek cakera adalah untuk memindahkan daya mekanikal dan menyebarkan tenaga haba yang dihasilkan akibat geseran semasa membrek dan keadaan suhu tinggi. Haba yang terlalu tinggi disebabkan oleh suhu tinggi semasa membrek akan menyebabkan ‘brake fade’, kehausan brek cakera, pengewapan cecair brek, kegagalan galas, brek cakera retak dan getaran akibat panas pada sistem brek cakera. Tujuan utama projek ini adalah untuk menentukan kontur suhu dan sifat tegasan brek cakera pada ‘DTDP ventilated disc brake’ dengan menggunakan perisian analisis unsur terhingga (FEA). Dalam projek ini, perisian analisis unsur terhingga MSC Nastran Patran 2005 digunakan untuk menganalisis suhu dan sifat tegasan pada brek cakera dalam tempoh masa membrek yang tertentu. ‘Ventilated disc brake’ model yang sedia ada akan dimodelkan dalam perisian CATIA V5R10 dan diimport kedalam perisian MSC Nastran Patran untuk kegunaan menganalisis. Sebelum memulakan kerja-kerja analisis, parameter penting seperti tenaga haba yang dikenakan pada permukaan membrek, pemalar bagi pemindahan haba perlakan pada permukaan membrek dan lubang pengudaraan akan ditentukan dengan kaedah matematik. Keputusan-keputusan kontur suhu, sudut anjakan yang disebabkan oleh pengembangan brek cakera akibat panas akan dikaji pada akhir simulasim proses. Keputusan daripada analisis ini menunjukkan keberkesanan lubang pengudaraan pada brek cakera dalam kapasiti penyejukan jika dibandingkan dengan brek cakera biasa.

ABSTRACT

The disc or rotor provides braking surface or friction surface for brake pads to rub against when braking is applied. The main function of brake rotor disc is for transmission of mechanical force and dissipation of heat produced implies to be functioning at both medium and high temperature. The excessive heat cause by high temperatures during braking may causes brake fade, premature wear, brake fluid vaporization, bearing failure, thermal cracks and thermally-excited vibration on disc brake systems. This project aims to determine the DTDP ventilated disc brake transient temperature and examine the disc thermal stress behaviour by using finite element analysis (FEA) software. In this project, the FEA software MSC NASTRAN 2005 is used to analyse the temperature and thermal stress behaviour of the disc under some periodic braking operation. An existing ventilated disc brake rotor model is then created in CATIA V5R10 program and is exported into MSC NASTRAN for analytical purposes. Before starting the analysis, an important parameter such as heat flux applied on the braking surface and convective heat transfer coefficient for braking surface and vane hole of disc is calculated via mathematical calculation. The results of temperature contours, coning angle cause by thermal expansion of disc are analyzed at the end of the simulation process. The result of analysis indicates that the vent holes show effectiveness for cooling capacity if compared to solid disc brake.

CONTENT

CHAPTER	TITLE	PAGE
	DECLARATION	ii
	DEDICATION	iii
	ACKNOWLEDGEMENT	iv
	ABSTRAK	v
	ABSTRACT	vi
	CONTENT	vii
	LIST OF TABLES	xi
	LIST OF FIGURES	xii
	NOMENCLATURES	xv
	LIST OF SYMBOLS	xvi
	LIST OF APPENDICES	xviii

CHAPTER I INTRODUCTION

1.1	Disc Brake Overview	1
1.2	Introduction	2
1.3	Problem Statement	3
1.4	Objective	3
1.5	Scope	4
1.6	Organization of Report	4

CHAPTER	TITLE	PAGE
CHAPTER II		LITERATURE REVIEW
2.1	Introduction	5
2.2	Review of Previous Research	6
2.3	Disc Brake or Rotor	8
2.3.1	Disc Brake or Rotor Properties	9
2.3.2	Aerodynamic of Ventilated Disc Brakes	10
2.3.3	Vane Geometry of Ventilated Disc Brake	10
2.3.4	Rotor Design and Operation Requirement	12
2.3.5	The Effects of Excessive Heat and Thermal Distortion on Disc or Drum Brake	14
2.3.5.1	Brake Fade	14
2.3.5.1.1	Lining Fade	14
2.3.5.1.2	Mechanical Fade	15
2.3.5.1.3	Gas Fade	15
2.3.5.2	Brake fluid vaporization	15
2.3.5.3	Thermal Crack	16
2.3.5.4	Brake Judder	16
2.4	Heat Transfer	17
2.4.1	Conduction	17
2.4.2	Convection	18
2.4.2.1	Natural convection	18
2.4.2.2	Forced convection	19
2.4.3	Radiation	20
2.4.4	Heat Transfer Coefficient	21
2.4.4.1	Heat Transfer Coefficient For Drum Brake	21
2.4.5.2	Heat Transfer Coefficient for Solid Disc	21
2.4.5.3	Heat Transfer Coefficient of Ventilated Disc Brakes	22

CHAPTER	TITLE	PAGE
2.5	Finite Element Analysis (FEA)	23
2.5.1	A Brief History of Finite Element Analysis (FEA)	23
2.5.2	FEA Applications	23
2.5.2.1	Pre-processing	24
2.5.2.2	Analysis (computation of solution)	24
2.5.2.3	Visualization (Post-Processing Phase)	24
2.5.3	Processes in Finite Element Analysis (FEA)	25
2.6	MSC Nastran	26
CHAPTER III		METHODOLOGY
3.1	Data Mining	27
3.1.1	Vehicle Information	27
3.1.2	Braking Schedule	29
3.2	Modeling	30
3.3	Formulation of The Analysis	30
3.3.1	Thermal Flow Evaluation	31
3.3.2	Boundary Condition: Convective Heat Transfer Coefficient	33
3.4	Simulation	38
3.4.1	The Finite Element Model	38
3.4.1.1	Method of Approach	39
3.4.2	Brake Disc Thermal Analysis	41
3.4.2.1	Analysis Assumption	42
3.4.2.2	Finite Element Analysis Stages	43
3.4.2.3	Periodic Operation	44
3.5	Analyse The Result	46
3.6	Methodology Flow Chart	47

CHAPTER	TITLE	PAGE
CHAPTER IV	RESULTS AND DISCUSSIONS	
4.1	Transient Temperature Results	49
4.2	Transient Temperature Discussions	60
4.3	Thermal Stress Results and Discussions	64
4.4	Disc Brake Deformation and Discussion	65
4.5	The Error Occur During Analysis	67
CHAPTER V	CONCLUSION AND RECOMMENDATION	
5.1	Conclusion	69
5.2	Recommendation for Future Work	70
REFERENCES		71
BIBLIOGRAPHY		73
APPENDIX		74

LIST OF TABLES

TABLE	TITLE	PAGE
3.1	Material Properties of Brake Disc	28
4.1	Time with Corresponding Temperature for First Braking Cycle	50
4.2	Time with Corresponding Temperature for Second Braking Cycle	51
4.3	Time with Corresponding Temperature for Third Braking Cycle	52
4.4	Time with Corresponding Temperature for Fourth Braking Cycle	53
4.5	Time with Corresponding Temperature for Fifth Braking Cycle	54
4.6	Time with Corresponding Temperature for Sixth Braking Cycle	55
4.7	Time with Corresponding Temperature for Seventh Braking Cycle	56
4.8	Time with Corresponding Temperature for Eighth Braking Cycle	57
4.9	Time with Corresponding Temperature for Ninth Braking Cycle	58
4.10	Time with Corresponding Temperature for Tenth Braking Cycle	59
4.11	Average Temperature Increase for Each Braking cycle	62

LIST OF FIGURES

FIGURE	TITLE	PAGE
1.1	Disc Brake Component	1
2.1	Brake Rotor Overview (Source: Owen 2004)	8
2.2	Air Flow on Ventilated Rotor (Source: Owen 2004)	8
2.3	SRV Rotor With Rounded Vane, SRV-R (Source: Wallis <i>et al.</i> 2002)	11
2.4	Diamond and Tear-Drop Pillars, DTDP Rotor (Source: Wallis <i>et al.</i> 2002)	11
2.5	Disc Thickness Variation (DTV) of A Rotor (Source: www.bendix.com.au/DTV.aspx)	13
2.6	Natural Convection, Inversion Layer (http://sol.sci.uop.edu/~jfalward/heattransfer/heattransfer.html)	19
2.7	Forced convection in automobile engine (http://sol.sci.uop.edu/~jfalward/heattransfer/heattransfer.html)	19
2.8	Process in FEA	25
3.1	Speeds versus Time in Braking Fading Procedure (Source: Gotowicki <i>et al.</i> 2005)	29
3.2	Chords Length of Inner Vane Passage	35
3.3	A_{in} Cross Section Area	36
3.4	A_{out} Cross Section Area	36
3.5	One Thirty Six Fraction Model Mesh	39
3.6	Full Disc Model Mesh	39

FIGURE	TITLE	PAGE
3.7	Axes-Symmetrical Surface	40
3.8	Disc Constrain Surfaces	40
3.9	Creating New Coordinate (Coord 1)	40
3.10	Creating Symmetrical Boundaries	40
3.11	The Energy Flow on Disc Model	41
3.12	The FEA Stages	43
3.13	Speed Profile for the First Braking	45
3.14	Inboard Side Heat Flux Profile	45
3.15	The Methodology Flow Chart of Project	47
4.1	Temperature Distribution Contour for First Braking Cycle (0 s – 32 s)	49
4.2	Temperature Distribution Contour for Second Braking Cycle (32 s – 64 s)	51
4.3	Temperature Distribution Contour for Third Braking Cycle (64 s – 96 s)	52
4.4	Temperature Distribution Contour for Fourth Braking Cycle (96 s – 128 s)	53
4.5	Temperature Distribution Contour for Fifth Braking Cycle (128 s – 160 s)	54
4.6	Temperature Distribution Contour for Sixth Braking Cycle (160 s – 192 s)	55
4.7	Temperature Distribution Contour for Seventh Braking Cycle (192 s – 224 s)	56
4.8	Temperature Distribution Contour for Eighth Braking Cycle (224 s – 256 s)	57
4.9	Temperature Distribution Contour for Ninth Braking Cycle (256 s – 288 s)	58
4.10	Temperature Distribution Contour for Tenth Braking Cycle (288 s – 320 s)	59

FIGURE	TITLE	PAGE
4.11	Temperature versus Time of Braking Surface	60
4.12	Temperature versus Time for Inner Vane Passage Surface	61
4.13	Temperature versus Time for Braking Surface and Vane Passage	62
4.14	Temperature Rise during First Braking	63
4.15	Temperature Rise during First Braking (Gotowicki et al. 2005)	63
4.16	Temperature Contour for Steady State Analysis	63
4.17	Thermal Stress Contour on Disc Rotor	64
4.18	Thermal Stress for Different View	64
4.19	Disc Rotor Deformation	65
4.20	Displacement Contour of the Disc Brake (X - Direction)	66
4.21	The Non-Spatial Time Function	67
4.22	The Input Data for Inboard Side	68

NOMENCLATURES

CAD	=	Computer-Aided Design
DTDP	=	Diamond and Tear Drop Pillars
DTV	=	Disc Thickness Variation
FEA	=	Finite Element Analysis
FEM	=	Finite Element Method
SRV	=	Straight Radial Vane
SRV-R	=	Straight Radial Vane with Rounded vane

LIST OF SYMBOLS

a	=	Deceleration, m/s^2
d_h	=	Rotor Passage Hydraulic Diameter, m
h_R	=	Convection Coefficient, $\text{W/m}^2\text{k}$
k_a	=	Air Thermal Conductivity, W/mk
L	=	Braking Work/ Energy
l	=	Cooling Vane Length, m
m	=	Vehicle Mass, kg
N_{disc}	=	Disc Rotational Speed, rpm
P_r	=	Prandtl Number = 0.72
Q	=	Heat Transfer, W
q	=	Thermal Flow, J/s
$q_{specific}$	=	Heat Flux, W/m^2
R_D	=	Disc Outer Radius, m
Re	=	Reynolds Number
R_{i-hub}	=	Disc Inner Radius of Rubbing Surface, m
r_{tyre}	=	Tyre Radius, m
s	=	Second
S_{flux}	=	Braking Surface, m^2
S_{in}	=	Chords Length, m
μ_a	=	Air Absolute Viscosity at 30°C = $1.86 \times 10^{-5} \text{ Ns/m}^2$
V	=	Vehicle Velocity, m/s
v	=	Air Kinematic Viscosity at 30°C = $1.60 \times 10^{-5} \text{ m}^2/\text{s}$

ω_{tyre}	=	Vehicle Angular Velocity, s^{-1}
ΔE	=	Kinetic Energy, J
Δt	=	Braking time, s
Δx	=	Stopping Distance, m

LIST OF APPENDICES

NO	TITLE	PAGE
A	The Subaru Impreza Data Sheet from Autocar Asean Magazine	75
B	Grey Cast Iron BS grade 220 Properties	76
C	The Existing DTDP Ventilated Disc Brake Geometry	78
D	Procedures for Analysis (MSC Nastran Patran)	80
E	Expected Result of Analysis	84
F	Gantt Chart (PSM I and PSM II)	85

CHAPTER 1.0

INTRODUCTION

1.1 Disc Brake Overview

The disc brake is a device for slowing or stopping the rotation of a wheel and to keep it from starting to move again. Disc brakes are widely used for reducing velocity for their characteristics of braking stability, controllability and the ability to provide a wide-ranging brake torque. Typical passenger vehicle, disc brake assembly consists of several major components namely disc or rotor, brake pad or friction material and caliper as shown in Figure 1.1. The disc or rotor is a flat round disc and usually made of cast iron or ceramic is attached to the wheel hub. Brake pad is positioned on opposite sides of the disc which is mounted on the brake caliper. Currently, there are two types of calipers which is fixed and floating or sliding caliper. Once the brake is applied, the brake pads are pushed by the hydraulic piston and squeezed against the disc surface and generates a friction force between the disc and pads in order to slow down or stop the vehicle (Owen, 2004).

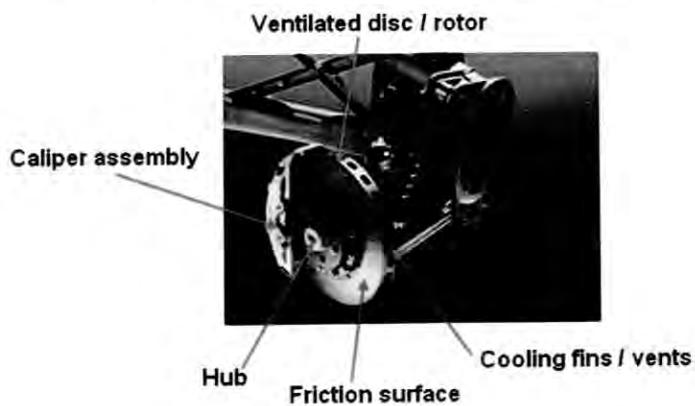


Figure 1.1: Disc Brake Component (Source: Adapted from Howstuffworks “How the Mercedes SLR Works”)

1.2 Introduction

An automotive brake is utilized to convert kinetic energy of motion into thermal energy through the friction between stationary brake pads and a rotating surface. Commonly this is done by the using a cast iron rotating brakes disc or rotor with stationary friction surfaces mounted in a caliper. The brake discs or rotors may be a solid discs or ventilated disc with two disc separated by a vane air flow passage. In bringing a vehicle to stop, the brake rotor must be able to store a significant amount of thermal energy since braking occurs in a relatively short period of time. Subsequently, the brake rotor must then dissipate the stored thermal energy quickly in order to be ready for the next application of the brake or friction contact. The ventilated brake rotors using an internal air passages for increase the level of forced convection and indirectly improve the rate of heat transfer. Cooling air is to be conveyed by the vanes enters the channels on the hub side at the inner circumference of the brake ring and exits at the outer circumference. The convective heat transfer coefficient of a vented disk brake rotor is approximately twice that of a solid disk (Limpert, 1972).

Most of the heat energy (99%) is dissipated through brake disc during the braking process. This major heat energy diffuses through conduction within the disc and through the hub and is dissipated by convection and radiation from the outer surface of disc and hub flanges. The temperature within the disc is transient in nature and is difficult to log correctly and fastly through convection method (Hwang *et al.* 2007).

The frictional heat generated on the interface of the rotor and the pad during braking can cause high temperature at the rotor surface which may deteriorate the material properties of the pad and causing the brake fade phenomena. The high temperature gradient at the surface may cause an excessive thermal stresses and causing in the surface crack after a long period of usage. Besides that, the uneven distribution of the temperature near the intersection of the hat and the rotor will result in thermal distortion of the frictional surface, which is know as coning and found to be a main cause of judder (Limpert, 1972). Therefore it is important to predict the

temperature rise and thermal deflection in the early design stage and for modification and improvement stage.

1.3 Problem Statement

Braking performance of a vehicle can be significantly affected by the temperature rise in the brake components. There are two main problem of disc brake which is related to the thermal, and is classified as excessive heat and severe thermal distortion. The excessive heat cause by high temperatures during braking may causes brake fade, premature wear, brake fluid vaporization, bearing failure, thermal cracks and thermally-excited vibration on disc brake system (Valvano and Lee 2000). For instance, the severe thermal distortion of a rotor can affect the important characteristics of brake such as reduce the system response and brake judder propensity. The above problems occur is mainly cause by the lacks of information about disc brake passenger vehicle performance and this is indirectly makes the disc brake designers fail to improve the existing disc construction.

1.4 Objective

- To determine the ventilated disc brake transient temperature contour and thermal stress behavior using FEA software (MSC NASTRAN PATRAN).
- By understanding the temperature transient behavior and contour of disc brake, it will become very helpful information for disc brake designer to improve the current disc brake problem and avoid disc brake from warp due to the excessive heat.

1.5 Scope

- The disc brake or rotor used in this analysis is DTDP passenger car ventilated disc brake.
- The brake periodic operation of braking sequence for total 320 seconds.

1.6 Organization of Report

The report is divided into several chapter/sections, corresponding to the following consideration. In Chapter 1, the introduction of project, problem statement, objective and scope of project is discussed. The complete temperature analysis is presented in Chapter 2, which states the literature of previous researches, types of discs or rotors, disc rotor properties and others.

The methodology to conduct this project is discussed in Chapter 3 whereby the method, modeling and mathematical calculations and simulations are presented. While in Chapter 4, the results of findings and discussions of results are presented. A brief summary and concluding remarks with recommendations are presented in Chapter 5.

CHAPTER 2.0

LITERATURE REVIEW

2.1 Introduction

The first patented model of disc brake was registered by Frederick William Lanchester in 1902. He described that a disc brake consists of a sheet of metal connected to one of the rear wheels of vehicle which is pinched at the wheel edge in order to slow down the vehicle. His invention leads towards the early sport type brake system design in the early 1950s which can be traced as development by Dunlop and the disc brake appeared in 1953 on the Jaguar C-Type racing car (Limpert, 1999). The sport-type design of disc brake is similar to the present type of disc brake which it can be found on any vehicle. However the materials used and actuation method used in the early period have been modified and improved.

Disc brakes offer better stopping performance than comparable drum brakes, including the resistance to "brake fade" caused by the overheating of brake components. Unlike a drum brake, the disc brake has no self-servo effect and the braking force is always proportional to the pressure placed on the braking pedal or lever. The most important contribution toward the widespread used of disc brake is due to the safety regulation throughout the world.

Disc brakes were most popular on sports cars when they were first introduced, since these vehicles are more demanding on brake performance. Discs brake now have become the more common form in most passenger vehicles, although drum brakes are many use on the rear wheels to keep costs and weight down as well as to simplify the provisions for a parking brake.