STUDY OF INTERNAL HEAT EXCHANGER EFFECTS ON CAR AIR CONDITIONING SYSTEM

CHAN HONG KET

This report is presented in partial fulfilment of the requirements for the Degree of Bachelor of Mechanical Engineering (Thermal Fluid)

> Faculty of Mechanical Engineering University Technical Malaysia Melaka

> > MAY 2010

C Universiti Teknikal Malaysia Melaka

VERIFICATION

I have read this thesis and from my opinion the thesis is sufficient in aspects of scope and quality for awarding Bachelor of Mechanical Engineering (Thermal Fluid)

Signature	:
Name of Supervisor	:
Date	:

DECLARATION

"I hereby, declare this thesis is the result of my own research except at cited in the reference"

Signature	:
Author's Name	: CHAN HONG KET
Date	:

PENGAKUAN

"Saya akui laporan ini adalah hasil kerja saya sendiri kecuali ringkasan dan petikan yang tiap-tiap satunya saya telah jelaskan sumbernya"

Tandatangan	:
Nama Penulis	: CHAN HONG KET
Tarikh	:



DEDICATION

To My Beloved Family

Mom & Dad

ACKNOWLEDGEMENT

I would like to express the deepest appreciation to my supervisor of this Final Year Project (*Projek Sarjana Muda*), Mr. Faizil bin Wasbari. Without his guidance and generously advices on my entire project; this Final Year Project report's accomplishment would not have been possible. Besides, I thank the Turbo machinery Laboratory technician Mr Razmi bin A. Razak for his knowledge and assist in the field of technical. In addition, a thank you to my helpful course mate, Miss Fasiha bt Nazirmuidin, who assisted me throughout the experiment section.

Special thank goes to Mr John J. Yarrish and Visteon Company for their sponsorship on the internal heat exchanger sample used in my experimental test. Besides, I appreciate the Faculty of Mechanical Engineering, University Teknikal Malaysia Melaka (UTeM) for giving opportunity and provide facilities for my project. Lastly, I would like to thank my friends whose have helped and give contribution for my project and report accomplishment.



ABSTRACT

Internal or suction line heat exchangers (IHX) are used in some car air conditioning systems with basic objective to optimalize the liquescence process of the refrigerant before entering the expansion device. This purpose is achieved by exchanging thermal energy between the cool gaseous refrigerant in the low pressured suction line and the warm liquid refrigerant in the high pressured liquid line in the A/C system. These devices can have either positive or negative influences on the Coefficient of Performance (COP) of the car air conditioning system, depending on the working fluids, the operating condition and the configuration of heat exchanger. The finding from the experimental test indicated the COP of the system with adoption of IHX will be up to 7.92% increment than the system without adoption of IHX if rotational speed of engine was increased.

ABSTRAK

Penukar haba dalaman atau saluran sedutan (IHX) selalunya digunakan dalam sistem penyamanan udara dengan objektif asalnya menyempurnakan proses pencecairan bahan penyejuk sebelum memasuki peralatan ekspansi. Tujuan tersebut dicapaikan dengan menukarkan tenaga haba di antara gas bahan penyejuk sejuk di salur sedutan yang tekanannya rendah dan cecair bahan penyejuk panas di salur cecair yang tekannya tinggi di dalam sistem penyamanan udara. Penukar haba boleh membawa impak positif atau negatif ke atas COP(Coefficient of Performance) sistem penyamanan udara kereta, bergantung dengan bahan penyejuk digunakan, syarat operasi, dan rekabentuk penukar haba. Kesimpulan dari kajian menyatakan bahawa COP sistem yang menggunakan penukar haba dalaman akan lebih tinggi sebanyak 7.92% daripada COP sistem yang tidak menggunakan penukar haba dalam sekiranya laju putaran enjin ditingkatkan.



TABLE OF CONTENTS

CHAPTER	CON	ITENTS	PAGES
	VER	IFICATION	ii
		LARATION	iii
		GAKUAN	iv
	DED	ICATION	V
	ACK	NOWLEDGEMENT	vi
	ABS	TRACT	vii
	ABS	TRAK	viii
	TAB	LE OF CONTENTS	ix
	LIST	COF TABLES	xvi
	LIST	COF FIGURES	XV
	LIST	COF SYMBOLS	xix
	LIST	COFAPPENDICES	XX
	NOMENCLATURE		xxi
CHAPTER I	INTI	RODUCTION	1
	1.1	Background Study	1
	1.2	Objectives	2
	1.3	Scope	2
	1.4	Problem Statement	2

CHAPTER	CON	TENTS		PAGES
CHAPTER II	LITE	RATUF	RE REVIEW	3
	2.1	Car Ai	r Conditioning System	3
		2.11	Compressor	6
		2.12	Condenser	7
		2.13	Expansion Valve	8
		2.14	Evaporator	9
		2.15	Receiver – drier	10
		2.16	Refrigerant Hose and Line	10
		2.17	Refrigerant R134a	11
	2.2	Vehicle	e Temperature Control System	12
	2.3	Ideal V	Vapour Compression Cycle	13
		2.31	Coefficient of Performance (COP)	14
	2.4	Interna	al Heat Exchanger	15
	2.5	Materi	al Selection	16
	2.6	Studie	s on Heat Transfer of IHX	20
		2.61	Fourier's Law of Conduction	20
		2.62	Newton's Law of Cooling	21
		2.63	The Overall Heat Transfer Coefficient	22
		2.64	The Log Mean Temperature Difference	e 24
		2.65	Effectiveness – NTU Method	25
	2.7	Evalua	ation of suction line/ liquid line heat	26
		exchar	nger on refrigeration cycle	
	2.8	8 Increase the evaporation temperature with the help of an internal heat exchanger		29
	2.9	Refrig	eration system performance using liquid	I - 31
		suction	n heat exchanger	
	2.10	Experi	mental evaluation of the internal heat	35
		exchar	nger influence on a vapour compression	

plant energy efficiency working with R22, R134a and R407.

CHAPTER III METHODOLOGY 38 3.1 Planning 38 Literature Study 3.2 40 3.3 40 Conceptual Design **Design Selection** 3.4 41 3.5 41 Material Selection 3.6 41 Preparation of Proposal 3.7 Material Purchase & Fabrication Process 42 **Experimental Setup** 3.8 42 3.9 Run Experiment 42 3.10 Results 43 3.11 43 Data Analysis 3.12 **Report Writing** 43

CHAPTER IVMODIFICATION444.1Purpose444.2Design Discussion444.3Modification of Parts45

CHAPTER V	EXPERIMENT		46
	5.1	The Experiment	46
	5.2	Preparation of Apparatus	46
	5.3	Testing Layout/ Setup	47

CHAPTER	CON	TENTS	PAGES
	5.4	Experiment Procedure	48
		5.4.1 System without IHX	48
		5.4.2 System with IHX	49
CHAPTER VI	RESULTS		
	6.1	Results for Configuration without IHX	51
	6.2	Results for Configuration with IHX	54
CHAPTER VII	DAT	AANALYSIS	56
	7.1	Heat Exchanger Effectiveness	56
	7.2	Refrigeration Cycle on P-h Diagram	58
	7.3	Fuel Consumption	62
CHAPTER VIII	DISC	CUSSION	63
	8.1	System Performance with Installation of IHX	K 63
	8.2	Limitation of Experiment	64
		8.2.1 Insulation Insufficiency	64
		8.2.2 Pressure Measurement	64
		8.2.3 Improper Modification of the IHX	64
		8.2.4 Neglected Parameters	65
		8.2.5 Different Evaporator & Condenser O	utlet
		Temperature of Systems	65
CHAPTER IX	CON	CLUSION & RECOMMENDATION	66

KIA	CUNCLUSION & RECOMMENDATION				
	9.1 Conclusion	66			



xii

CHAPTER	CONTENTS	PAGES
	9.2 Recommendation	67
	REFERENCES	69
	APPENDICES	71



LIST OF TABLES

NO	TITLE	PAGES
2.1	Properties table of age-hardening wrought Aluminium (Source: CES Selector Version 4.6, Granta Design Limited)	16
2.2	Properties table of Copper (Source: CES Selector Version 4.6, Granta Design Limited)	17
2.3	Properties table of Brass (Source: CES Selector Version 4.6, Granta Design Limited)	18
6.1	Result for 1000 rpm without IHX	51
6.2	Result for 1500 rpm without IHX	52
6.3	Result for 2000 rpm without IHX	52
6.4	Result for 2500 rpm without IHX	53
6.5	Result for 1000 rpm with IHX	54
6.6	Result for 1500 rpm with IHX	54
6.7	Result for 2000 rpm with IHX	55
6.8	Result for 2500 rpm with IHX	55
7.1	Effectiveness of the IHX	56
7.2	Comparison of the performance with and without IHX	60

LIST OF FIGURES

NO	TITLE	PAGES
2.1	Automotive air conditioning system (Source: pakwheels.com (2009))	3
2.2	Temperature and Pressure on A/C system Compressor and its sectional view (Source: ensautoservice.com (2009))	4
2.3	Compressor and its sectional view (Source: www.ensautoservice.com (2009))	6
2.4	Condenser (Source: autocyd.en.made-in-china.com (2009))	7
2.5	Thermal expansion valve and its sectional view (Source: allproducts.com (2009))	8
2.6	Evaporator (Source: tradeget.com (2009))	9
2.7	Typical Barrier Hoses (Source: sovereign-publications.com (2009))	10

2.8	Refrigerant R134a and Its Properties (Source: wikipedia.com (2009))	11
2.9	Air flow in passenger compartment (Source: sovereign-publications.com (2009))	12
2.10	Refrigeration cycle of refrigerant R134a on p-h diagram (Source: nzifst.org.nz (2009))	13
2.11	Internal Heat Exchangers developed by Denso (<i>left</i>) and Hydro (<i>right</i>) (Source: denso.com & hydro.com (2009))	15
2.12	Counter-flow double pipe heat exchanger	22
2.13	Temperature distribution in parallel (<i>right</i>) and counter-flow (<i>left</i>) HX (Source: tpub.com (2009))	24
2.14	Change of the COP for different effectiveness of LSHX at $T_e = -23^{\circ}C$ and $T_c = 42.5^{\circ}C$ (Source: Domanski P.A. (1994))	27
2.15	Simulation of R134a performance (Source: Domanski P.A. (1994))	27
2.16	Schematic of IHX (5) installation (Source: Tambovtsev A. (Unknown year))	29
2.17	p-h diagram of idealized effect of SLHX (Source: Klein et al (2000))	31

2.18	RCI as function of SLHX effectiveness ignoring correction for system mass flow changes. (Source: Klein et al (2000))	32
2.19	Correction to the COP to account for pressure loss in the low pressure leg of the LSHX. (Source: Klein et al (2000))	34
2.20	The Packless HRX-250A (tube-in-tube) Internal Heat Exchanger (Source: Navarro-Esbri et al. (2003))	35
2.21	Relative pressure drops comparison at suction line with and without IHX (Source: Navarro-Esbri et al. (2003))	36
2.22	Relative variation of the refrigerating capacity due to IHX adoption (Source: Navarro-Esbri et al. (2003))	37
3.1	Flow Chart of Methodology	39
4.1	The internal heat exchanger before modification	44
4.2	The parts of IHX needed to be modified	45
4.3	The internal heat exchanger after modification	45
5.1	Schematic diagram of experiment setup without IHX	47
5.2	Schematic diagram of experiment setup with IHX	47

5.3	Testing layout without IHX	48
5.4	Testing layout with IHX	49
7.1	IHX effectiveness change as function of the engine rotational speed	57
7.2	Refrigeration cycle of rational speed 1000rpm	58
7.3	Refrigeration cycle of rational speed 1500rpm	59
7.4	Refrigeration cycle of rational speed 2000rpm	59
7.5	Refrigeration cycle of rational speed 2500rpm	60
7.6	COP as a function of the rotational speed	61
7.7:	Fuel consumption as a function of the rotational speed	6.2



LIST OF SYMBOLS

COP	=	Coefficient of Performance
c _p	=	Specific heat capacity at constant pressure, kJ/kg K
h	=	Enthalpy, kJ/kg
ṁ	=	Mass flow rate
n	=	Polytrophic index
P _c	=	Compressor electrical power consumption, kW
Р	=	Pressure, kPa
q_{o}	=	Refrigerating effect, kJ/kg
Qo	=	Refrigerating capacity, kW
t	=	Compression ratio
Т	=	Temperature, °C or K
v	=	Specific volume, m ³ / kg
IHX	=	Internal heat exchanger
SLHX	=	Suction-line heat exchanger
3	=	Effectiveness
Δp	=	Pressure drop, kPa
ΔT	=	Temperature difference, °C or K

Subscripts

g	=	Gaseous
Κ	=	Condenser
1	=	Liquid
0	=	Evaporator
R	=	Refrigeration

LIST OF APPENDICES

NO	TITLE	PAGES
A	Gantt chart of PSM 1 & 2	71
В	Refrigerant Temperature Pressure Table	74
С	CES Edupack 2005 Snapshot	76
D	IHX Technical Drawing	79
Е	Miscellaneous	81



NOMENCLATURE

Temperature - A measure of the average kinetic energy of the particles in a sample of matter, expressed in terms of units or degrees designated on a standard scale.

Degree Celsius - A temperature scale (also called *centigrade*) in which 0° represents freezing and 100° represents the boiling point. To convert Celsius temperatures to Fahrenheit scale, multiply the Celsius figure by 9, divide by 5 and add 32.

Absolute Temperature - The reading on a scale with zero at the thermodynamic null, most particularly the Kelvin scale.

Pressure - It is defined as force per unit area. It is usually more convenient to use pressure rather than force to describe the influences upon fluid behaviour. The standard unit for pressure is the Pascal, which is a Newton per square meter.

Specific volume - The volume occupied by a unit of mass of a material. It is equal to the inverse of density.

Heat - The energy which is spontaneously flowing from an object with a high temperature to an object with a lower temperature.

Conduction - The transfer of thermal energy between the neighbouring molecules in a substance due to temperature gradient.

Convection – The process by which heat is transferred by movement of a heated fluid such as air or water.



Radiation - The process by which heat is transferred between bodies by electromagnetic radiation.

Latent heat – the characteristic amount of energy absorbed or released by a substance during a change in its physical state that occurs without changing its temperature.

Specific heat - the measure of the heat energy required to increase the temperature of a unit quantity of a substance by unit degree.

Critical Temperature – The maximum point at which a gas can be liquefied or condensed by raising the pressure.

Critical Pressure – The pressure that is necessary to liquefy a gas at that temperature.

Phases of Refrigerant

a) *Compressed liquid* – Pure liquid, at less than saturation temperature (boiling point at pressure)

b) *Saturated liquid* – Pure liquid, but at the saturation temperature (any additional of heat will cause some vaporization)

c) *Saturated liquid/ vapour mixture* – A mixture of liquid and vapour at the temperature and pressure of saturation.

d) *Saturated vapour* – Purely vapour, but at the saturation temperature (any loss of heat will cause some condensation to occur)

e) Superheated vapour – Purely vapour, above the saturation temperature.

Subcritical System – The refrigeration system where condenser and evaporator operate with temperature below the refrigerant's critical temperature.

Transcritical System – The refrigeration system where the cycle incurs temperatures and pressures both above and below the refrigerant's levels.



Humidity – the amount of water in the ambient air. The amount of water vapour suspended in air can vary from a perfectly dry 0% to foggy 100%.

Relative Humidity (RH) – The amount of water vapour air hold varies with temperature.

CHAPTER I

INTRODUCTION

1.1 Background Study

Internal heat exchanger (IHX) also known as Suction-line/Liquid-line heat exchanger (SLHX) is a derivation from the traditional heat exchangers. Its purpose/concept is similar with other type of heat exchangers, which is a device to build for efficient transfer heat from one place to another. However, it has several notable differences with other traditional heat exchanger in term of size, shape, structure, usage, etc.

Nowadays, the developments of internal heat exchanger are rapid. It has an important role on air-conditioning system based on the vapour compression cycle. The internal heat exchanger often use in automotive air conditioning, with the basic objective of assuring the subcooling of refrigerant before entering the expansion device. This process is achieved by exchanging enthalpy (energy) between the cool gaseous refrigerant in the suction line and the hot liquid refrigerant at liquid line. However, according to previous research result, the devices can have positive or negative influences on the overall efficiency, depending on it design and configuration.