BALANCING EQUIPMENT FOR HIGH RPM PULLEY

ABDUL HAKAM BIN SYED SYAMSUDIN

UNIVERSITI TEKNIKAL MALAYSIA MELAKA

C Universiti Teknikal Malaysia Melaka

BALANCING EQUIPMENT FOR HIGH RPM PULLEY

ABDUL HAKAM BIN SYED SYAMSUDIN

This report is submitted in fulfilment of the requirement for the degree of Bachelor of Mechanical Engineering (Plant and Maintenance)

Faculty of Mechanical Engineering

UNIVERSITI TEKNIKAL MALAYSIA MELAKA

2017



DECLARATION

I declare that this report entitled "*Balancing Equipment for High RPM Pulley*" is the result of my own research except summaries and quotations which have been acknowledged. The report has not been accepted for any other degree and is not concurrently submitted in candidature of any other degree.

Signature	:	
Name	:	Abdul Hakam Bin Syed Syamsudin
Date	:	

C Universiti Teknikal Malaysia Melaka

APPROVAL

I hereby declare that I have read this project report and in my opinion this report is sufficient in terms of scope and quality for the award of the degree of Bachelor of Mechanical Engineering (Plant and Maintenance).

Signature	:	
Name of Supervisor	:	Dr. Reduan Mat Dan
Date	:	



DEDICATION

I would like to dedicate to

My father,

SYED SYAMSUDIN BIN SYED AHMED

My mother,

ZAHARAH BT AHMAD

My supervisor,

DR. REDUAN MAT DAN

and

All my friend,

for their assistances & supportive efforts.

C Universiti Teknikal Malaysia Melaka

ABSTRACT

Unbalanced mechanical systems are always be the problem to engineers because of its impact can distrupt the smoothness and the reliability of a system. From the past, unbalance is known as the most causes of machine vibration and cause more vibration and generates excessive force in bearing area and will reduces the life of machine used. In oder to expert and understand the unbalance characteristics of these pulley, a pulley balancer equipment that can stand untill to 3000 RPM speed of motor must be designed in this project. In balancing the pulley, method of mass addition and vibration measurement technique are used. The design must consist the suitable material and strong to it stand untill the maximum speed. The result outcome the increasing of speed untill to 300 RPM without vibrate the pulley and unbalanced the pulley. As conclusion, the pulley must be designed by using the right method, requirement and properly assemble. Therefore helps in monitoring the health of rotating component where could be useful to prevent the breakdown.

ABSTRAK

Ketidakseimbangan sistem mekanikal sentiasa menjadi masalah kepada para jurutera kerana kesannya boleh menganggu kelancaran kebolehpercayaan sistem itu. Dari masa lalu, ketidakseimbangan dikenali sebagai punca utama getaran mesin dan menyebabkan lebih banyak getaran dan menghasilkan daya yang berlebihan di kawasan galas dan akan mengurangkan kehidupan mesin yang digunakan. Untuk pakar dan memahami ciri-ciri ketidakseimbangan takal ini, peralatan pengimbang takal yang bole bertahan sehingga 3000 RPM kelajuan motor mestilah direka dalam projek ini. Dalam mengimbangi takal, kaedah penambahan berat dan teknik pengukuran getaran digunakan. Reka bentuk mestilah terdiri daripada bahan yang sesuai dan kuat untuk bertahan sehingga kelajuan maksimum. Hasilnya adalah peningkatan kelajuan sehingga 3000 RPM tanpa gegaran takal dan ketidakseimbangan takal. Sebagai kesimpulan, takal mesti direka dengan menggunakan kaedah, keperluan dan pemasangan yang betul. Selain daripada itu, membantu dalam pemantauan kesihatan komponen berputar di mana berguna untuk mencegah pecahan.

ACKNOWLEDGEMENT

In the name of Allah, most Gracious, most Compassionate. I would like to express my full gratitude to Allah for granting me the best condition of health and strength in completing my final year project. Special thanks to my supervisor, Dr. Reduan Mat Dan for his moral supports, commitment to the students and continuous guidance in helping completing my final year project in these two semesters. Besides that, special appreciation to the technical staff Puan Hidayah for his commitment in helping by using the laboratory equipment. I also would like to appreciate to all my friends for giving me a full assistance and additional knowledge in helping me to complete my research. Not to forget my beloved mother and father who have encouraged me during their life to learn until this high levels and also to my beloved family for their understanding the time I to sacrifice in this semester and also previous semester for my research. Thank you for their priceless moral support that they have given to me. Last but not least, appreciation to Universiti Teknikal Malaysia Melaka for giving the chance to me to complete my undergraduate program and gaining tonnes of priceless knowledge.

TABLE OF CONTENTS

	PAGE
ABSTRACT	i
ABSTRAK	ii
ACKNOWLEDGEMENT	iii
LIST OF FIGURES	viii
LIST OF TABLES	xi
LIST OF ABBEREVATIONS	xiii
LIST OF SYMBOL	xiv
CHAPTER 1	1
INTRODUCTION	1
1.1 BACKGROUND	1
1.2 PROBLEM STATEMENT	3
1.3 OBJECTIVE	3
1.4 SCOPE OF PROJECT	3
CHAPTER 2	4
LITERATURE REVIEW	4
2.1 THEORY OF BALANCING	4
2.2 TYPES OF UNBALANCE	5
2.2.1 Static unbalance	5
2.2.2 Couple unbalance	7

2.2.3 Dynamic unbalance	9
2.3 FORMATION OF EQUATIONS AND ANALYSIS	10
2.4 PULLEY AND BELT SYSTEM	15
2.4.1 Forces between belt and pulley	17
2.5 PREVIOUS STUDY	20
2.5.1 Balancing a system using variable speed drivers	20
2.5.2 Load torque problem	20
2.5.3 Cyclic Load Factor (CLF) approach	21
2.5.4 Discrete Fourier Transform (DFT) approach	22
2.5.5 Result using CLF method	23
2.5.4 Result using DFT method	24

CHAI	CHAPTER 3			
METI	HODOI	LOGY	26	
	3.1 INTRODUCTION3.2.EXPERIMENTAL SETUP			
 3.1 INTRODUCTION 3.2.EXPERIMENTAL SETUP 3.2.1 Schematic diagram 3.2.2 Function of part 				
		3.2.1 Schematic diagram	28	
		3.2.2 Function of part	30	
	3.3	FABRICATION OF PULLEY	31	
		3.3.1 Spectrum frequency analysis	33	
	3.4	BALANCING METHOD	35	
		3.4.1 Vector Method	35	
		3.4.2 Flow chart	37	

CHAPTER 4	38
RESULT AND DISCUSSION	38
4.1 INTRODUCTION	38
4.2.RESONANCE	40
4.2.1 10hz-50hz	42
4.2.2 40hz-50hz	43
.4.3 BASELINE	45
4.2.1 MOTOR ONLY	45
4.2.2 WITHOUT ANY PULLEY	46
4.2.3 SINGLE PULLEY (center shaft)	47
4.2.4 SINGLE PULLEY (at inboard bearing)	48
4.2.5 SINGLE PULLEY (at outboard bearing)	49
4.4 UNBALANCED	50
4.4.1 SINGLE PULLEY (center shaft)	50
4.4.2 SINGLE PULLEY (at inboard bearing)	51
4.4.2.1 Effect of adding mass (7 g)	51
4.4.3 SINGLE PULLEY (at outboard bearing)	52
4.4.3.1 Effect of adding mass (7 g)	52
4.5 BALANCING	53
4.5.1 SINGLE PULLEY (center shaft)	53
4.5.2 SINGLE PULLEY (at inboard bearing)	56
4.5.3 SINGLE PULLEY (at outboard bearing)	59

CHAPTER 5	62
CONCLUSION AND RECOMMENDATION	62
REFERENCESS	64
APPENDIX A	67
APPENDIX B	71
APPENDIX C	73

LIST OF FIGURES

NO	TITLE	PAGE
2.1	Static unbalance	6
2.2	Couple unbalance	7
2.3	Static unbalance and couple unbalance	8
2.4	Dynamic unbalance	9
2.5	Parametric transformation associated with the solid in rotation	11
2.6	Geometric definition of the position of balancing masses	14
2.7	Simple pulley system	15
2.8	Pulley wheel	16
2.9	Velocity ratio	16
2.10	Diagram of abrasive belt on driven pulley	17
2.11	Variable speed driver	25
3.1	Machinery Fault Simulator	27
3.2	Machine Fault Simulator operation (schematic diagram)	28
3.3	Drawing of pulley to use at MFS machine	31
3.4	Unbalance (static) spectrum frequency	33
3.5	Unbalance (vertical) spectrum frequency	33
	viii	

C Universiti Teknikal Malaysia Melaka

3.6	Unbalance (overhug) spectrum frequency	34
3.7	Vector Plot Diagram	35
3.8	Balancing vector plot for determination balanced mass and position	36
3.9	Flowchart of Methodology	37
4.1	Front view of pulley that has been fabricated	39
4.2	Side view of pulley that been fabricated	39
4.3	Graph of amplitude against frequency of motor from 10 Hz to 50 Hz	40
4.4	Graph of amplitude againts frequency of motor from 40 Hz to 50 Hz	42
4.5	Figure of motor only	45
4.6	Schematic diagram of experiment runs without any pulley	46
4.7	Schematic diagram of experiment runs with pulley at center of shaft	47
4.8	Schematic diagram of experiment runs with pulley at inboard bearing	48
4.9	Schematic diagram of experiment runs with pulley at outboard bearing	49
4.10	Schematic diagram of experiment runs with unbalance pulley at center	50
	shaft	
4.11	Schematic diagram of experiment runs with unbalance pulley at inboard	51
	bearing	
4.12	Schematic diagram of experiment runs with unbalance pulley at outboard	52
	bearing	
4.13	Schematic diagram of experiment runs with balance pulley at center shaft	53

4.14	Graph of amplitude before balancing and after balancing pulley at center	53
	shaft	
4.15	Schematic diagram of experiment runs with balance pulley at inboard	56
	bearing	
4.16	Graph of amplitude before balancing and after balancing pulley at	56
	inboard bearing	
4.17	Schematic diagram of experiment runs with balance pulley at outboard	59
	bearing	
4.18	Graph of amplitude before balancing and after balancing pulley at	59
	outboard bearing	

LIST OF TABLES

NO	TITLE	PAGE
2.1	Values of CLF calculated unit with balanced load and unbalanced load	23
	with 8kg and 16kg	
2.2	Values of the first harmonic with DFT calculated for the unit with balanced	24
	load and imbalancedload using 8kg and 16kg	
3.1	Data Measurement Setup	29
4.1	Results of amplitude of pulley from 10 Hz to 50 Hz	40
4.2	Results of amplitude of pulley from 40 Hz to 50 Hz	43
4.3	Table of amplitude of vibration for motor only	45
4.4	Results of baseline in inboard and outboard bearing	46
4.5	Results of baseline both bearing with pulley at center of shaft	47
4.6	Results of baseline both bearing with pulley at inboard bearing	48
4.7	Results of baseline both bearing with pulley at outboard bearing	49
4.8	Results of amplitude both bearing with unbalance pulley at center shaft	50
4.9	Results of amplitude both bearing with unbalance pulley at inboard bearing	51
4.10	Results of amplitude both bearing with unbalance pulley at outboard bearing	52
4.11	Results balancing with balance pulley at center shaft in vector method	54

4.12	Results balancing	g with balance	pulley at inbo	ard bearing in ve	ector method	57
------	-------------------	----------------	----------------	-------------------	--------------	----

4.13 Results balancing with balance pulley at outboard bearing in vector method 60

LIST OF ABBEREVATIONS

- RPM Rotation Per Minute
- MFS Machinery Fault Simulator
- VSD Variacle Speed Driver
- CLF Cyclic Load Factor
- DFT Discrete Fourier Transform
- HZ Hertz

LIST OF SYMBOL

T= belt tesion

- p = average normal pressure between belt and pulley
- p_2 = average normal pressure at slack side end of belt

R= radius of pulley

- θ = angle of contact
- α = active angle
- τ = average shear stress

$A_a = real contact area$

- A= apparent contact area
- μ = coefficient of friction
- c, a, B= constant

CHAPTER 1

INTRODUCTION

1.1 BACKGROUND

Unbalance in rotating machinering has been found to be one of the most common causes of machinery vibration. In any industry, rotating machinery is a basic part. Unbalance may develop in the system due to the operating conditions such as manufacturing, assembly, installed machines and other causes. Fabrication problems for example distorted castings, offbeat machining and poor gathering can also create unbalance. As example distortion problems is rotational stresses, aerodynamic and temperature change. Many of these occur during manufacture and others during operational existence of machine.

In addition, improper assembly is the reason why unbalance occurs when a rotor is being fabricated. As a theory, when a unbalanced shaft and unbalanced rotor united, a radial diplacement occurs from the necessary assembly which will produce an unbalance condition. These such as include consumption of the machine, wear and less right connection. The large unbalances will cause require large weight corrections and this can have negative impact on the integrity of the rotor.

The driven pulley and belt also must be studied. Forces is produced from normal and tangential and friction characteristics between an abrasive belt and pulley. Then, suitable size and material that can be used to withstand until 3000 RPM speed. As a theory, classical Euler equation state that for a flat belt power transmission assumes a constant coefficient of friction

between pulley and belt. It also state that the coefficient of friction between rubber structure and hard surface depends to normal pressure, material consistent and shear stress. (Kim,H., 1987)

There are many equipment that are used to verify the imbalance system. The equipment will be used based on to check either the system is unbalance or not. One of the equipment is using Machinery Fault Simulator (MFS) and Vabbit pro vibration machinery diagnostic system by analyzing generated frequency at spectrum. The equipment can detect faults in gears, bearings and other mechanical components. A parameter-free method to analyze sensor signals that incorporates two or more frequency demodulation, phase demodulation of the raw signal data and amplitude demodulation. Any of equipment will use a sensors to detect the unbalance condition.

One example of sensors is an accelerometer. This device can measures a proper acceleration and have multiple applications in science or industry. It can detect and monitor vibration in rotating machinery and has a single or multi-axis to detect direction and magnitude of proper acceleration. Accelerometer works in many ways, two from it are capacitance sensor and piezoelectric effect. The capacitance accelerometer senses changes in between microstructures located next to the device and if a force moves, the capacitance will change to voltage for interpretation. For the piezoelectric effect, it is the most common form accelerometer and uses microscopic crystal structures that become accelerative forces. (Natalia, 2013)

1.2 PROBLEM STATEMENT

As known, the pulley is one of the important component in many machinery and industry users. However, the manufacturing is not perfect and there will be some defects occurred. Unbalance pulley will result in vibration that can affect bearing and many others component. In addition, the vibration sensor might not detect at very low rpm, but if the speed increase to 3000 RPM, the unbalance might become significant. This project would fabricate and test unbalance pulley with the speed of 3000 RPM.

1.3 OBJECTIVE

The objectives of this project are as follows :

- 1. To investigate unbalance pulley at various speed.
- 2. To do balancing on pulley using vector method.

1.4 SCOPE OF PROJECT

The scopes of this project are:

- The cheapest way to design pulley balancing equipment with 3000 RPM speed of rotation motor.
- 2. Result of balancing testing to it stand until 3000 RPM speed of motor.

CHAPTER 2

LITERATURE REVIEW

2.1 THEORY OF BALANCING

Unbalance in the general definition is the combination between the "dynamic" unbalance and "static" unbalance (Krysinski, T., & Malburet, F, 2007). It also one of the conventional vibratory sources in rotating systems. The mass that circulate in rotating parts around the axis of rotation may generate inertial effects in specific cases. These will create vibrations in cyclic loads in the links and bearer structure involved. These loads generally noticeable and it is important for the structure to rotate at high speeds or when the structure is rotate there has a large mass with an inappropriate mass contribution around the axis of rotation. In mechanical systems rotating machinery is commonly used, including industrial turbomachinery, machining tools and etc. Vibration caused by mass unbalance is a normal problem in rotating machinery. Unbalance occurs if the geometric axis is not coincident with the principal axis of inertia of the rotor.

The center of inertia is on the axis of rotation and the axis of rotation is a principal axis of inertia must be to sure so that the equilibrium is obtain and to avoid unbalance. The systems are in equilibrium when the masses of rotating elements are distributed equitably around the axis of rotation and the resultant inertial effects are zero. The vibration and noise will not happened when a machine is in equilibrium. For a proper dynamic operation of machines these two parameters are important to be considered. A slight of asymmetry in rotating parts is enough to create an unbalance that causes dynamic responses at the bearings for high rotation speeds. Higher rotation speeds also can cause much greater centrifugal unbalance forces and current pattern of rotating equipment toward higher operational speeds to higher power density openly leads.

2.2 TYPES OF UNBALANCE

There are three types of unbalance :

- i. Static unbalance
- ii. Couple unbalance
- iii. Dynamic unbalance

2.2.1 Static unbalance

Defined as the eccentricity of the center of gravity (MacCamhaoil, 1989) of a rotor that caused when the center of rotation has point mass at a certain radius. It also defines when the rotor will roll and it stops when its heavy spot is at the lowest position and has *in-phase* motion between both end of the rotor. It required to restore from the center of gravity to the center of rotation when an equal mass is placed at angle of 180° to the unbalanced mass and at the same radius. The static balancing including settling primary forces into one plane and adding a correction mass into that plane only. Many rotating parts that have most their concentrated mass in or very near one plane, such as car wheels, flywheels or etc can be determined as static balancing problems.